

Mixed-Integer Nonlinear Optimization for District Heating Network Expansion

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Trier University

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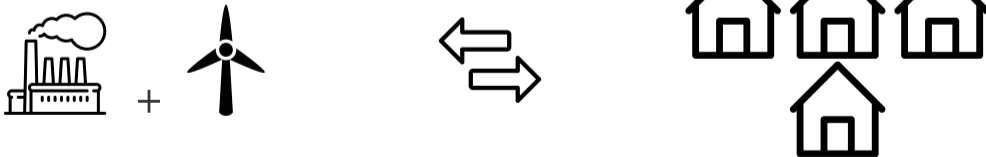
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⇒ District Heating Networks

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Optimization

- Control (Benonysson et al. 1995; Krug et al. 2020; Sandou et al. 2005; Verrilli et al. 2017)
- Design (Ameri and Besharati 2016; Blommaert et al. 2018; Bracco et al. 2013; Guelpa et al. 2018; Mertz et al. 2017; Söderman 2007)
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Challenges

- Nonlinear behaviour of fluids in pipes
- Mixing of water at pipe intersections
- Water flow directions
- Modeling of **candidate** consumers

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Assumptions

- One dimensional stationary model
- Only tree shaped networks

Prologue

Modeling

Problem Formulation

Network and Physics Modeling

Thermal Energy Equation Approximation

Candidate Pipe Modeling

Numerical Results

Test Case

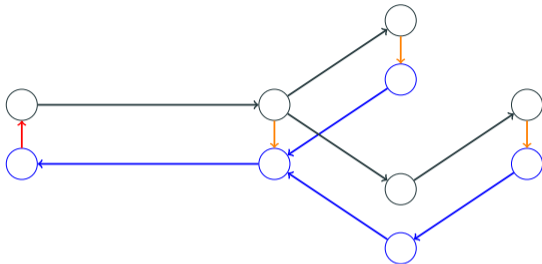
Initial Results

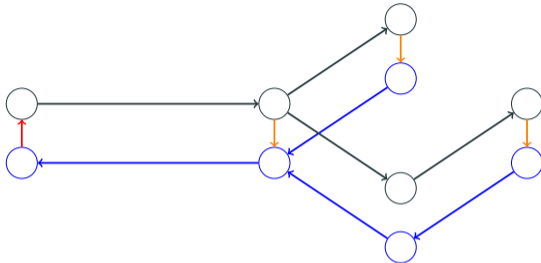
Discussion

Conclusion

max Profit = Consumer Payment - Expansion, Operation and Maintenance Cost
s.t. System physics
Consumer constraints
Depot constraints

Network Modeling and Notation

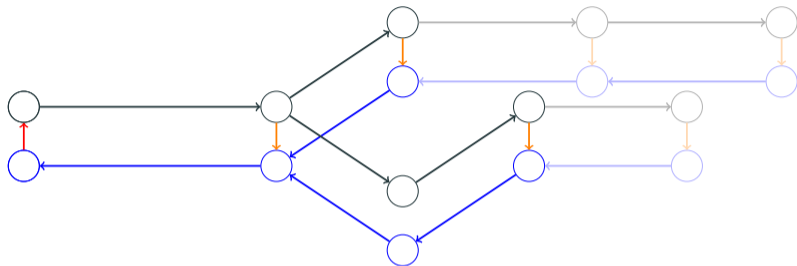




We introduce the graph $G = (V, A)$ such that

- A_{ff}, A_{bf}, A_c, a_d
- V_{ff}, V_{bf}

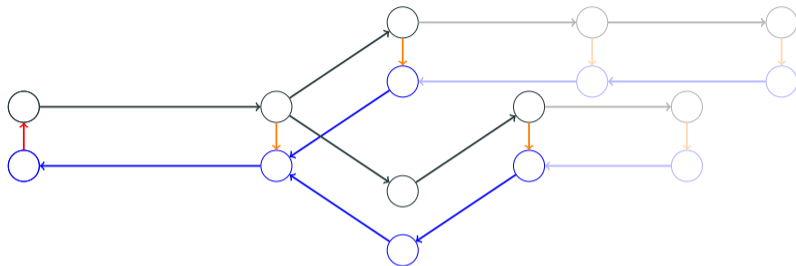
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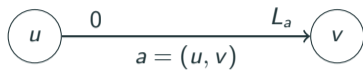
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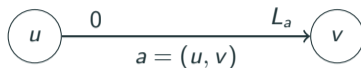
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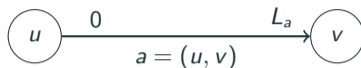
- A_{ff}, A_{bf}, A_c, a_d
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- $A_{ff}^e, A_{ff}^c, A_{bf}^e, A_{bf}^c, A_c^e, A_c^c$
- $V_{ff}^e, V_{ff}^c, V_{bf}^e, V_{bf}^c$





One-dimensional stationary momentum equation for compressible fluids in cylindrical pipes
(Hauschild et al. 2020; Krug et al. 2020)

$$\frac{p_a(L_a) - p_a(0)}{L_a} = -g\rho h'_a - \lambda_a \frac{|v_a| v_a \rho}{2D_a}, \quad \forall a \in A_{\text{ff}} \cup A_{\text{bf}}$$



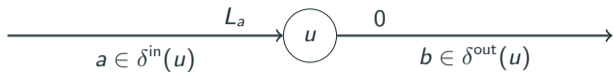
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One-dimensional stationary thermal energy equation

$$T_a(L_a; v_a) = \begin{cases} T_{\text{soil}}, & v_a = 0 \\ (T_a(0) - T_{\text{soil}}) e^{-\frac{4U_a L_a}{c_p \rho D_a v_a}} + T_{\text{soil}}, & v_a > 0 \end{cases}, \quad \forall a \in A_{\text{ff}} \cup A_{\text{bf}}$$





Mass flow continuity

$$\sum_{a \in \delta^{\text{in}}(u)} q_a = \sum_{a \in \delta^{\text{out}}(u)} q_a, \quad u \in V$$



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Pressure continuity

$$p_u = p_a(0), \quad u \in V, \quad a \in \delta^{\text{out}}(u)$$

$$p_u = p_a(L_a), \quad u \in V, \quad a \in \delta^{\text{in}}(u)$$



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Temperature mixing (Krug et al. 2020)

$$T_u = \frac{\sum_{a \in \delta^{\text{in}}(u)} c_p q_a T_a(L_a)}{\sum_{a \in \delta^{\text{in}}(u)} c_p q_a}, \quad u \in V$$

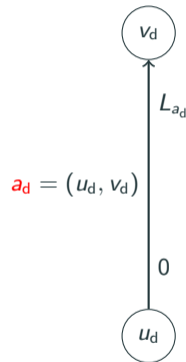
$$T_u = T_a(0), \quad u \in V, \quad a \in \delta^{\text{out}}(u)$$

For $\mathbf{a}_d = (u_d, v_d)$

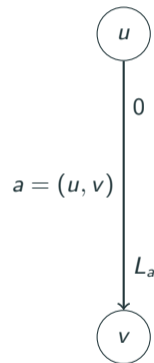
$$p_{u_d} = p_s$$

$$P_p = \frac{q_{a_d}}{\rho} (p_{v_d} - p_{u_d})$$

$$P_w + P_g = q_{a_d} c_p (T_a(L_{a_d}) - T_a(0))$$



$$\begin{aligned}
 P_a &= q_a c_p (T_a(0) - T_a(L_a)), & a \in A_c^e \\
 x_a P_a &= q_a c_p (T_a(0) - T_a(L_a)), & a \in A_c^c \\
 T_a(L_a) &= T^{bf}, & a \in A_c^e \cup A_c^c \\
 T_a(0) &\geq T_a^{ff}, & a \in A_c^e \cup A_c^c \\
 p_v &\leq p_u, & a \in A_c^e \cup A_c^c
 \end{aligned}$$



Thermal Energy Equation Approximation I

$$f(v_a, T_a(0), T_a(L_a)) := \begin{cases} (T_a(0) - T_{\text{soil}}) e^{-\frac{4U_a L_a}{c_p \rho D_a v_a}} + T_{\text{soil}} - T_a(L_a), & v_a > 0 \\ T_{\text{soil}} - T_a(L_a), & v_a = 0 \end{cases}$$

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⇓

$$f_{\text{approx}}(v_a, T_a(0), T_a(L_a)) := \sum_{(k,l,m) \in \Theta_d} \alpha_{klm} v_a^k T_a(0)^l T_a(L_a)^m + T_a(L_a) - T_{\text{soil}},$$

with

$$\Theta_d := \left\{ (k, l, m) \in \mathbb{N}^3 : k \neq 0 \text{ and } k + l + m \leq d \right\}.$$

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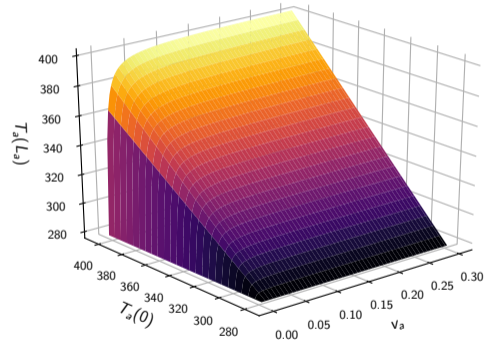
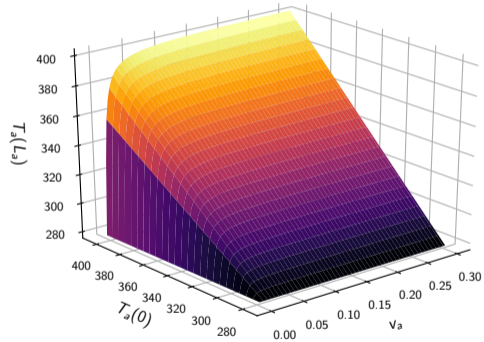
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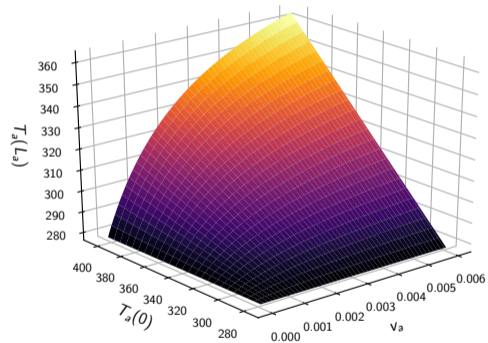
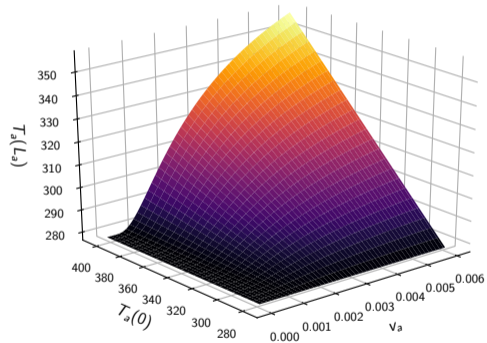


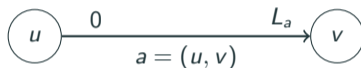
$$\min_{\alpha} \sum_{i \in I} f_{\text{approx}}(v_a^i, T_a(0)^i, T_a(L_a)^i)^2$$

Thermal Energy Equation Approximation II



Thermal Energy Equation Approximation III

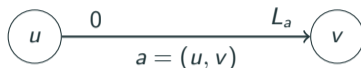




One-dimensional stationary momentum equation for compressible fluids in cylindrical pipes

$$p_v - p_u + L_a g \rho h'_a + \lambda_a \frac{|v_a| v_a \rho L_a}{2D_a} = 0, \quad \forall a \in A_{\text{ff}} \cup A_{\text{bf}}$$

Candidate Pipe Modeling: Pressure Constraints



One-dimensional stationary momentum equation for compressible fluids in cylindrical pipes

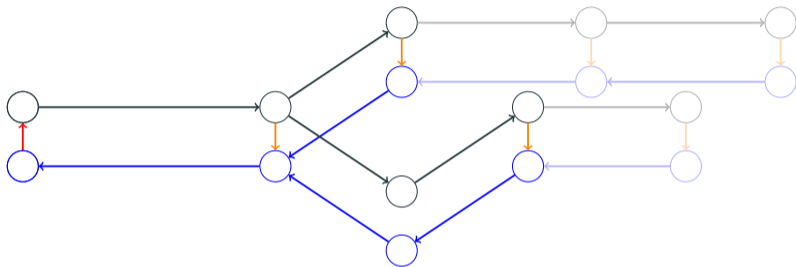
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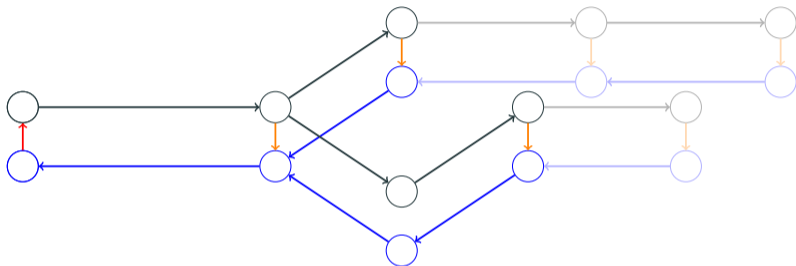
$$p_v - p_u + L_a g \rho h'_a + \lambda_a \frac{|v_a| v_a \rho L_a}{2D_a} \leq (1 - x_a) M_a^1, \quad \forall a = (u, v) \in A_{ff}^c \cup A_{bf}^c$$

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Candidate Pipe Modeling: Valid Inequalities



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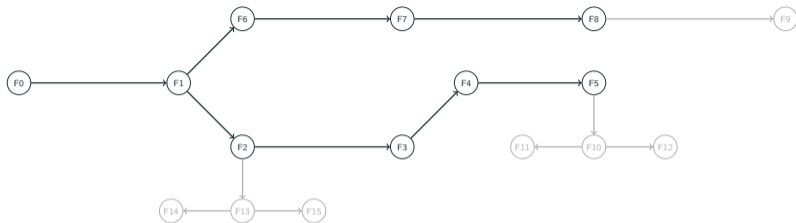
$$x_{a_1} \leq x_{a_2}, \quad a_1, a_2 \in A_{\text{ff}}^c \cup A_{\text{bf}}^c \cup A_c^c, \quad a_2 \in P(a_1)$$

Objective

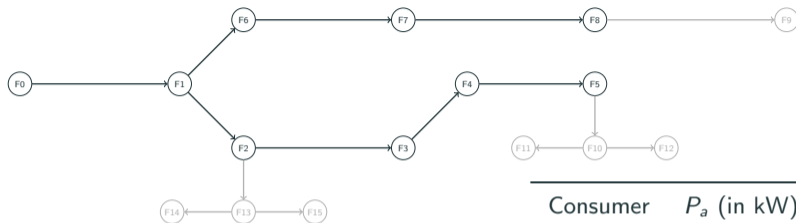
$$\max \sum_{a \in A_{\xi}^c} P_a w \pi x_a - \sum_{a \in A_{ff}^c \cup A_{bf}^c \cup A_{\xi}^c} C_a^{\text{inv}} x_a - w (C_p P_p + C_w P_w + C_g P_g)$$

max objective

s.t. stationary incompressible Euler equation
stationary thermal energy equation approximation
mass conservation
pressure continuity,
temperature mixing equations
depot constraints
consumer constraints
system bounds
power bounds
valid binary inequalities



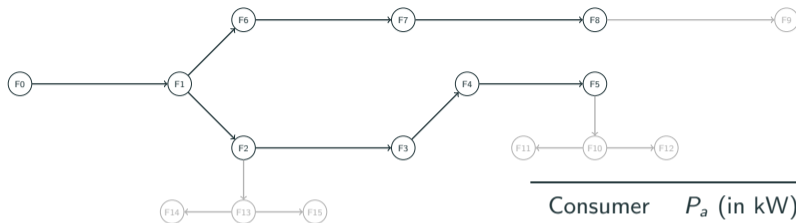
Test Case



Cost	C_w	C_g	C_p
Value (€/kWh)	0	0.0415	0.165
Edge	$a \in A_c^c$	$a \in A_{ff}^c \cup A_{bf}^c$	
C_a^{inv} (€)	100 000	[90 000, 330 000]	

Consumer	P_a (in kW)
(F2,B2)	200.00
(F3,B3)	600.00
(F5,B5)	150.00
(F6,B6)	666.66
(F8,B8)	200.00
(F9,B9)	183.33
(F11,B11)	183.33
(F12,B12)	183.33
(F14,B14)	183.33
(F15,B15)	183.33

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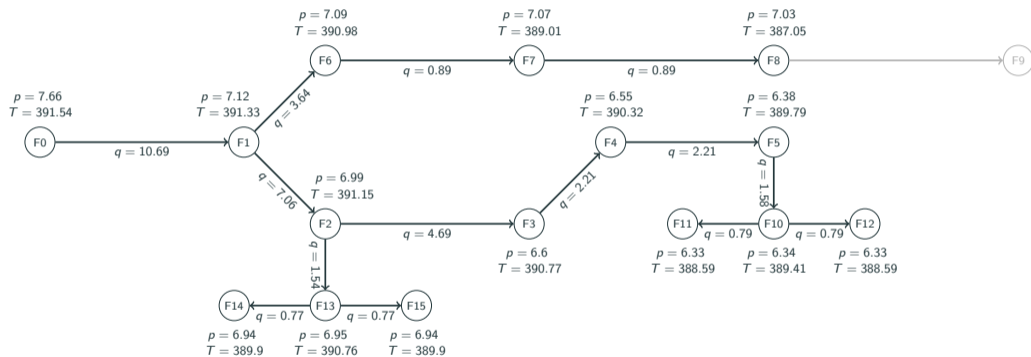


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Intel(R) Core(TM) i7-8550U, 16 GB RAM.
 ANTIGONE using the Pyomo interface.

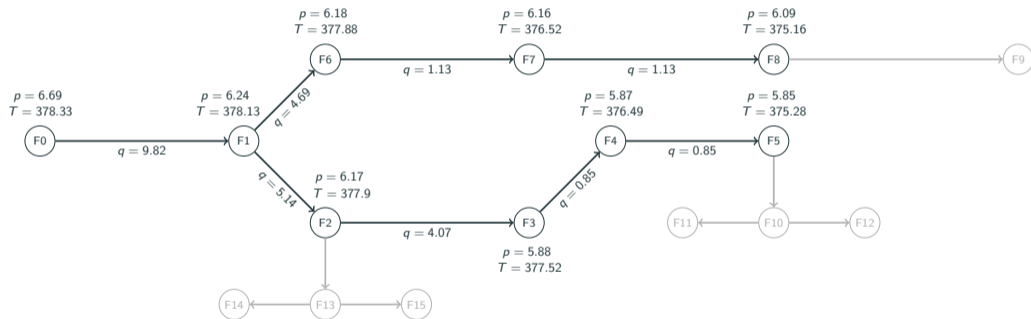
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Initial Results



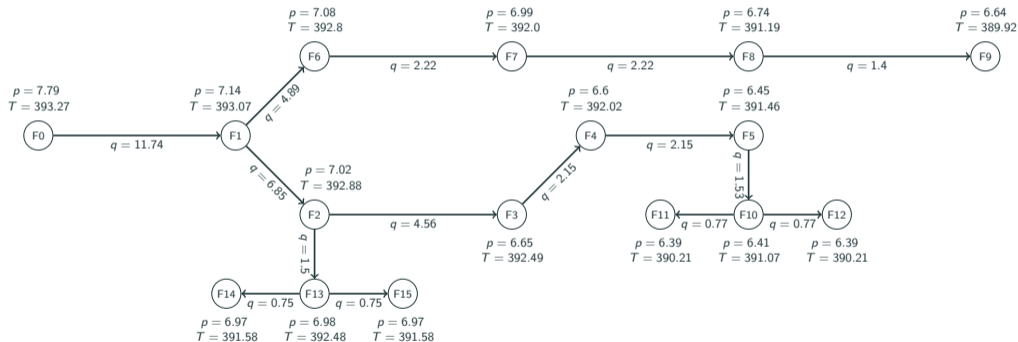
Discussion: Impact of Estimated Average Demand

If $P_a = 150$ kW for all $a \in A_c^e$



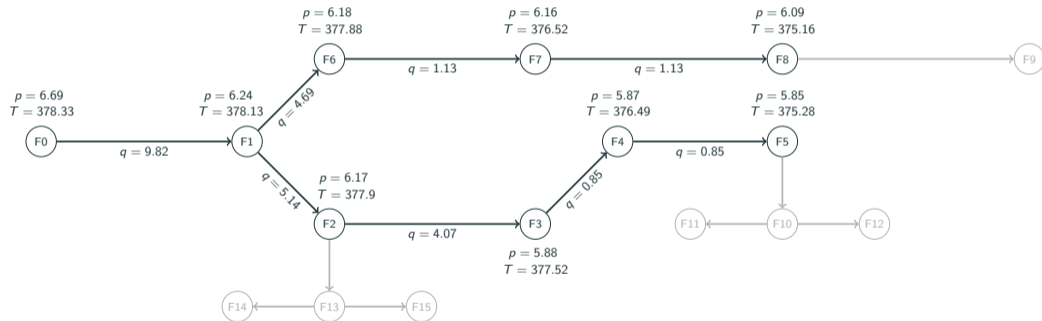
Discussion: Impact of Distance

If $P_a = 350$ kW for all $a \in A_c^c$



Discussion: Impact of Thermal Losses

If $U_a = 0.5 \text{ W m}^{-1} \text{ K}^{-2} \Rightarrow 0.8 \text{ W m}^{-1} \text{ K}^{-2}$



Contributions of the expansion model

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- Nonlinear modeling of T and p behavior

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- Polynomial approximation of the thermal energy equation

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Main parameters

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